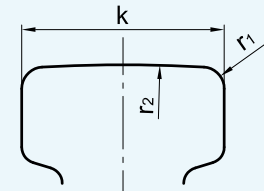


Table 1. **Symbol and unit**

symbol	unit	description	explanation
$c_1$	-	material coefficient	Values in accordance with table 2
$c_2$	-	speed coefficient	Values in accordance with table 3a and 3b
$c_3$	-	operating time coefficient	Values in accordance with table 4
$d_1$	mm	Travelling wheel diameter	Running surface diameter
$n$	min <sup>-1</sup>	Speed of crane wheel	Values in accordance with table 3b
$p$	N/mm <sup>2</sup>	Pressure	$p = \frac{R}{c_2 \cdot c_3 \cdot d_1 (k - 2r_1)}$
$p_{zul}$	N/mm <sup>2</sup>	Permissible pressure between crane wheel and rail	$p_{zul} = 5,6 c_1$
$k$	mm	Rail head width	 <p>For cambered crane rails the ideal effective rail head width will be <math>k - 2r_1</math>.</p>
$r_1$	mm	Radius of curvature of rail head	
$r_2$	mm	Radius of camber of rail head	
$k - 2r_1$	mm	Ideal effective rail head width	Values for crane rails in accordance with table 5
$v$	m/min	Speed of crane wheel	
$R$	N	Wheel force	For crane travelling wheels $R = \frac{R_{min} + 2R_{max}}{3}$ For trolley travelling wheels $R = R_{max}$
$R_{max}$	N	Maximum wheel force	$R_{max}$ and $R_{min}$ should be determined from the most frequent operating positions of the loaded trolley
$R_{min}$	N	Minimum wheel force	
$R_0$	N	Characteristic wheel force	Values in accordance with table 6

## Calculation of crane rail wheels

The wheel force is calculated using the formula:

$$R \leq p_{zul} \cdot c_2 \cdot c_3 \cdot d_1 \cdot (k - 2r_1) \quad (1)$$

From the above is obtained the crane wheel diameter

$$d_1 \geq \frac{R}{p_{zul} \cdot c_2 \cdot c_3 \cdot (k - 2r_1)} \quad (2)$$

The characteristic wheel force  $R_0$  is obtained from equation (1), where:

$$\begin{aligned} p_{zul} &= 5,6 \text{ N/mm}^2 \\ c_2 &= 1 \\ c_3 &= 1 \end{aligned}$$

are applied for  $R_0 = 5,6 \cdot d_1 \cdot (k - 2r_1)$  (3)

When using the characteristic wheel force the permissible wheel force can be calculated in simplified fashion using the formula:

$$R \leq R_0 \cdot c_1 \cdot c_2 \cdot c_3 \quad (4)$$

## Rail/crane wheel material matching

Table 2. **Permissible pressure  $p_{zul}$  and material coefficient  $c_1$**

material minimum tensile strength [N/mm <sup>2</sup> ]		$p_{zul}$	$c_1$
rail	wheel	[N/mm <sup>2</sup> ]	
590	£ 330	2,8	0,50
	410	3,6	0,63
	490	4,5	0,80
	590	5,6	1,00
≥ 690	≥ 740	7,0	1,25
	≥ 800	7,2	1,29
	≥ 900	7,8	1,39
≥ 700	≥ 1000	8,5	1,52

The hardening of the running surfaces with a depth of  $0,01 \cdot \text{diameter}$  can be considered selecting  $p_{zul}$ .

Table 3a. **speed coefficient  $c_2$**

wheel-Ø $d_1$	$c_2$														
	for v in m/min														
	10	12,5	16	20	25	31,5	40	50	63	80	100	125	160	200	250
<b>200</b>	1,09	1,06	1,03	1	0,97	0,94	0,91	0,87	0,82	0,77	0,72	0,66	-	-	-
<b>250</b>	1,11	1,09	1,06	1,03	1	0,97	0,94	0,91	0,87	0,82	0,77	0,72	0,66	-	-
<b>315</b>	1,13	1,11	1,09	1,06	1,03	1	0,97	0,94	0,91	0,87	0,82	0,77	0,72	0,66	-
<b>400</b>	1,14	1,13	1,11	1,09	1,06	1,03	1	0,97	0,94	0,91	0,87	0,82	0,77	0,72	0,66
<b>500</b>	1,15	1,14	1,13	1,11	1,09	1,06	1,03	1	0,97	0,94	0,91	0,87	0,82	0,77	0,72
<b>630</b>	1,17	1,15	1,14	1,13	1,11	1,09	1,06	1,03	1	0,97	0,94	0,91	0,87	0,82	0,77
<b>710</b>	-	1,16	1,14	1,13	1,12	1,1	1,07	1,04	1,02	0,99	0,96	0,92	0,89	0,84	0,79
<b>800</b>	-	1,16	1,15	1,14	1,13	1,11	1,09	1,06	1,03	1	0,97	0,94	0,91	0,87	0,82
<b>900</b>	-	-	1,16	1,14	1,13	1,12	1,1	1,07	1,04	1,02	0,99	0,96	0,92	0,89	0,84
<b>1000</b>	-	-	1,17	1,15	1,14	1,13	1,11	1,09	1,06	1,03	1	0,97	0,94	0,91	0,87
<b>1100</b>	-	-	-	1,16	1,14	1,13	1,12	1,1	1,07	1,04	1,02	0,99	0,96	0,92	0,89
<b>1250</b>	-	-	-	1,17	1,15	1,14	1,13	1,11	1,09	1,06	1,03	1	0,97	0,94	0,91

Tabelle 3b.

**wheel speed n from  
speed coefficient  $c_2$**

$c_2$	$n^a$ [min <sup>-1</sup> ]
0,66	200
0,72	160
0,77	125
0,79	112
0,82	100
0,84	90
0,87	80
0,89	71
0,91	63
0,92	56
0,94	50
0,96	45
0,97	40
0,99	35,5
1	31,5
1,02	28
1,03	25
1,04	22,4
1,06	20
1,07	18
1,09	16
1,1	14
1,11	12,5
1,12	11,2
1,13	10
1,14	8
1,15	6,3
1,16	5,6
1,17	5

Table 4. **operating time coefficient  $c_3$**

operating time of travelling gear (referred to 1 hour)	$c_3$
bis 16%	1,25
über 16 bis 25%	1,12
über 25 bis 40%	1
über 40 bis 63%	0,9
über 63%	0,8

Tabelle 5. **ideal effective rail head width ( $k-2r_1$ )**

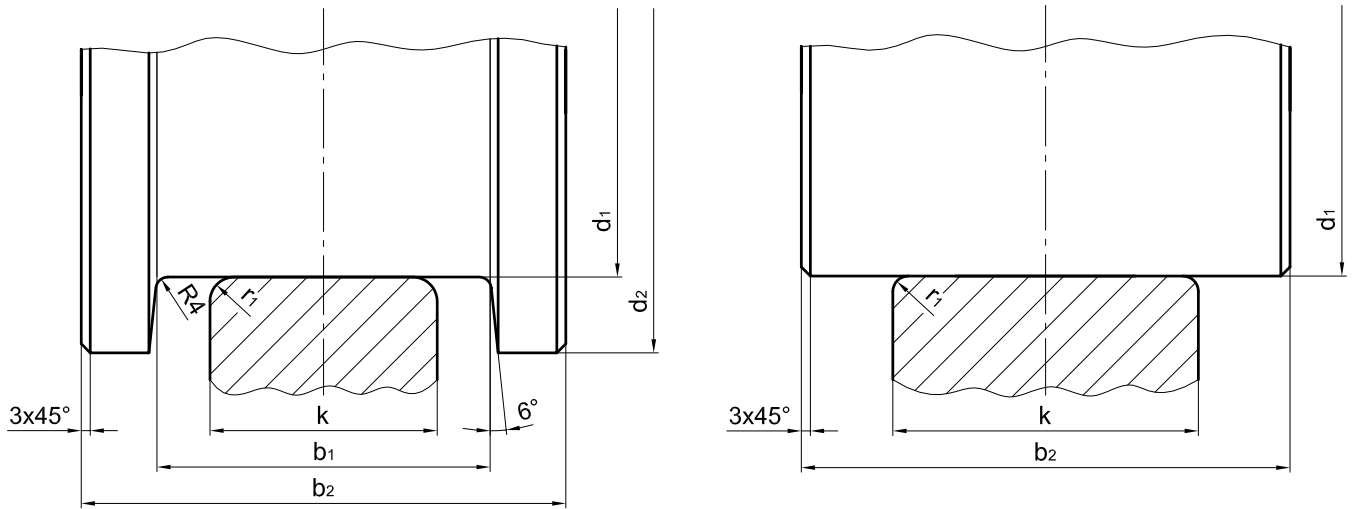
as per	crane rails		$r_1$ mm	$k-2r_1$ mm	
	DIN	designation			
		new			previous
536 Teil 1	A 45	KS 22	4	37	
	A 55	KS 32	5	45	
	A 65	KS 43	6	53	
	A 75	KS 56	8	59	
	A 100	KS 75	10	80	
	A 120	KS 101	10	100	
536 Teil 2	F 100	-	5	90	
	F 120	-	5	110	

Tabelle 6. **characteristic wheel force  $R_0$**

wheel-Ø $d_1$	$R_0$ in N for narrow wheels				$R_0$ in N for broad wheels					$R_0$ in N for wheels without wheelflange	
	for crane rail				for crane rail					for crane rail	
	A 45	A 55	A 65	A 75	A 55	A 65	A 75	A 100	A 120	F 100	F 120
<b>200</b>	41000	50000	-	-	-	-	-	-	-	-	-
<b>250</b>	52000	63000	-	-	-	-	-	-	-	-	-
<b>315</b>	65000	79000	-	-	79000	93000	-	-	-	-	-
<b>400</b>	83000	101000	-	-	101000	119000	132000	-	-	202000	-
<b>500</b>	104000	126000	-	-	126000	148000	165000	-	-	252000	-
<b>630</b>	-	159000	187000	-	-	187000	208000	282000	-	318000	388000
<b>710</b>	-	178000	211000	235000	-	-	235000	318000	398000	358000	437000
<b>800</b>	-	201000	237000	264000	-	-	264000	358000	448000	403000	493000
<b>900</b>	-	-	267000	297000	-	-	297000	403000	504000	454000	554000
<b>1000</b>	-	-	297000	330000	-	-	330000	448000	560000	504000	616000
<b>1120</b>	-	-	-	-	-	-	-	502000	627000	-	-
<b>1250</b>	-	-	-	-	-	-	-	560000	700000	-	-

# Running surface profiles of crane wheels and correlation of crane rails to wheel-diameter

DIN 15 072



Crane wheels with wheel flange

Crane wheels without wheel flange

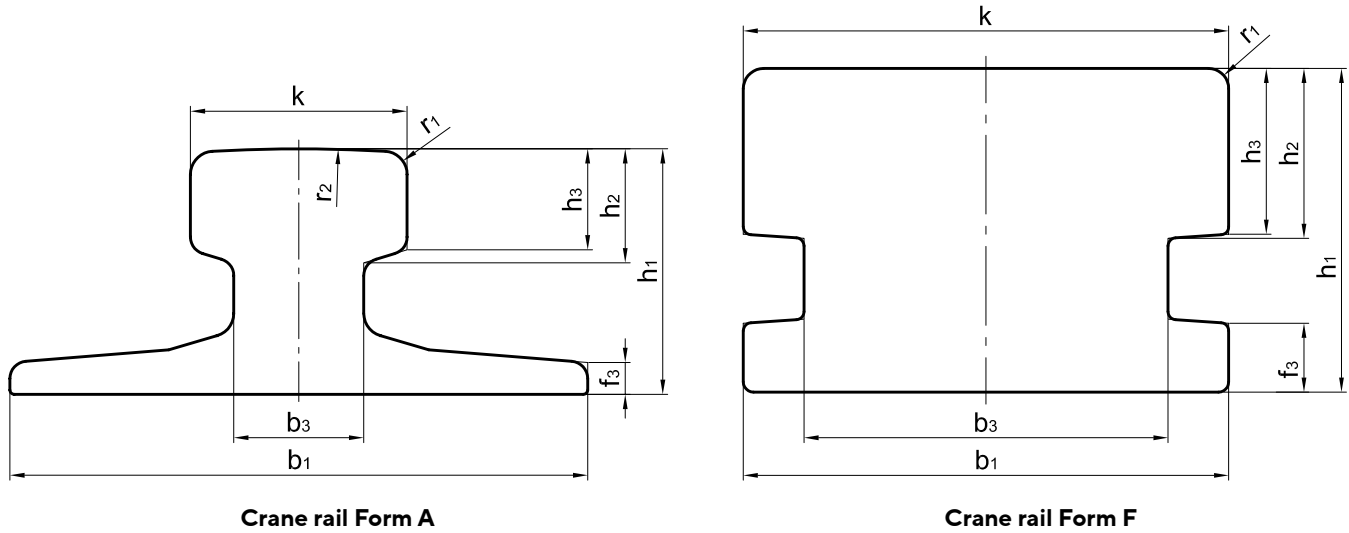
Crane wheel-Ø $d_1$	$d_2$	for crane wheels with narrow wheel flange						for crane wheels with broad wheel flange						for crane wheels without wheel flange				
		for crane rail <sup>1)</sup>				$b_1$	$b_2$	for crane rail <sup>1)</sup>					$b_1$	$b_2$	for crane rail <sup>2)</sup>		$b_2$	
		A 45	A 55	A 65	A 75			A 55	A 65	A 75	A 100	A 120			F 100	F 120		
h9		k				max.		k					max.		k			
<b>200</b>	230	45	-	-	-	55	90	-	-	-	-	-	-	-	-	-	-	-
<b>250</b>	280	45	-	-	-	55	90	-	-	-	-	-	-	-	-	-	-	-
<b>315</b>	350	45	-	-	-	55	90	55	-	-	-	-	65	110	-	-	-	
<b>400</b>	440	45	55	-	-	65	110	55	65	75	-	-	90	140	100	-	140	
<b>500</b>	540	45	55	-	-	65	110	55	65	75	-	-	90	140	100	-	140	
<b>630</b>	680	-	55	65	-	75	120	-	65	75	100	-	110	160	100	120	160	
<b>710</b>	760	-	-	65	75	90	140	-	-	75	100	120	160	210	100	120	210	
<b>800</b>	850	-	-	65	75	90	140	-	-	75	100	120	160	210	100	120	210	
<b>900</b>	950	-	-	65	75	90	140	-	-	75	100	120	160	210	-	120	210	
<b>1000</b>	1050	-	-	65	75	90	140	-	-	75	100	120	160	210	-	120	210	
<b>1120</b>	1180	-	-	-	-	-	-	-	-	-	100	120	160	220	-	-	-	
<b>1250</b>	1310	-	-	-	-	-	-	-	-	-	100	120	160	220	-	-	-	
$r_1$		4	5	6	8	-	-	5	6	8	10	10	-	-	5	5	-	

1) Crane rail acc. to DIN 536-1.

2) Crane rail acc. to DIN 536-2.

# Champignon rail acc. to DIN 536

main dimensions for information, dimensions can vary depending on the producer



**Crane rail Form A**

**Crane rail Form F**

nominal size	k	b <sub>1</sub>	b <sub>3</sub>	h <sub>1</sub>	h <sub>2</sub>	h <sub>3</sub>	f <sub>3</sub>	r <sub>1</sub>	r <sub>2</sub>	ideal effective rail head width $k - 2r_1$ (acc. to DIN 15070)
<b>A 45</b>	45	125	24	55	24	20	8	4	400	37
<b>A 55</b>	55	150	31	65	28,5	25	9	5	400	45
<b>A 65</b>	65	175	38	75	34	30	10	6	400	53
<b>A 75</b>	75	200	45	85	39,5	35	11	8	500	59
<b>A 100</b>	100	200	60	95	45,5	40	12	10	500	80
<b>A 120</b>	120	220	72	105	55,5	47,5	14	10	600	100
<b>A 150</b>	150	220	80	150	64,5	50	14	10	800	130
<b>F 100</b>	100	100	70	80	42	41	17	5	-	90
<b>F 120</b>	120	120	90	80	42	41	17	5	-	110